

ME 343 – Control Systems

Lecture 02
August 26, 2009

What is Control for?

Feedback control can help:

- reference following (tracking)
- equilibrium regulation
- disturbance rejection
- stability
- performance

} CHANGING DYNAMIC BEHAVIOR

First step in this design process: DYNAMIC MODEL

Model Classification

Model Representation $\longleftrightarrow$ **Control Technique**

Spatial Dependence	Lumped parameter system $f = f(t)$ Ordinary Diff. Eq. (ODE)	Distributed parameter system $f = f(t, x)$ Partial Diff. Eq. (PDE)
Linearity	Linear	Nonlinear
Temporal Representation	Continuous-time	Discrete-time
Domain Representation	Time	Frequency

Spatial Dependence

Distributed Parameter Systems

PDE $n_e = n_e(t, r)$

$$\frac{\partial n_e}{\partial t} = -\frac{1}{r} \frac{\partial}{\partial r} r \left(D_n \frac{\partial n_e}{\partial r} - n_e V_n \right) + S_n(t, r)$$

$$\left. \frac{\partial n_e}{\partial r} \right|_{r=0} = 0 \quad n_e|_{r=a} = n_e^a$$

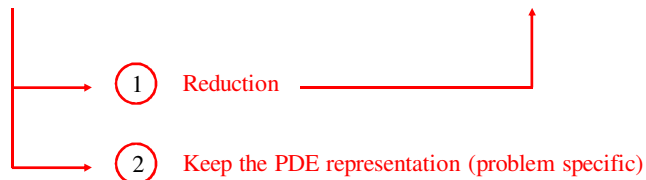
Lumped Parameter Systems

ODE $n_e = n_e(t)$

$$\frac{dn_e}{dt} = -\frac{1}{\tau_e} n_e + S_n(t)$$

$$n_e(0) = n_{e0}$$

Control: **Interior** **Boundary**



Linearity: **Nonlinear/Linear**

Linear/Nonlinear Distributed Parameter Control

Linear/Nonlinear Lumped Parameter Control

Linearity

Nonlinear (ODE) Systems

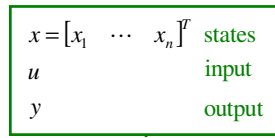
$$y^{(n)} = g(t, y^{(n-1)}, \dots, y, u^{(m-1)}, \dots, u)$$

Non-Autonomous

$$\begin{aligned}\dot{x} &= f(t, x, u) \\ y &= h(t, x, u)\end{aligned}$$

Autonomous

$$\begin{aligned}\dot{x} &= f(x, u) \\ y &= h(x, u)\end{aligned}$$



Linear (ODE) Systems

$$y^{(n)} + a_{n-1}y^{(n-1)} + \dots + a_0y = b_{m-1}u^{(m-1)} + \dots + b_0u$$

$$\begin{aligned}\dot{x} &= A(t)x + B(t)u \\ y &= C(t)x + D(t)u\end{aligned}$$

LTV

$$\begin{aligned}\dot{x} &= Ax + Bu \\ y &= Cx + Du\end{aligned}$$

LTI

Nonlinear Control

1 Linearization

2 Keep the nonlinearities

Output/State Feedback

Estimation: How to estimate states from input/output?

Linear Control

Linearity

Particular type of nonlinearities: Constraints

LTI

$$\begin{aligned}\dot{x} &= Ax + Bu \\ y &= Cx + Du\end{aligned}$$

$$\begin{aligned}u, y &\rightarrow \text{sat}(u), \text{sat}(y) \\ \underline{x}_i &< x_i < \bar{x}_i\end{aligned}$$

input/output constraints
state constraints

1 A priori

Constraint is considered for control design

2 A posteriori

Constraint is NOT considered for control design

Anti-windup Techniques

Temporal Representation

Continuous-time Systems

Discrete-time Systems

$$y^{(n)} + a_{n-1}y^{(n-1)} + \dots + a_0y = b_{m-1}u^{(m-1)} + \dots + b_0u$$

Sampled-Data Systems

$$y[kT] + a_{k-1}y[(k-1)T] + \dots + a_ny[(k-n)T] = b_0u[kT] + \dots + b_my[(k-m)T]$$

Sampling Time

LTI

$$\begin{aligned} \dot{x}(t) &= Ax(t) + Bu(t) \\ y(t) &= Cx(t) + Du(t) \end{aligned}$$

$$\begin{aligned} x[k+1] &= Ax[k] + Bu[k] \\ y[k] &= Cx[k] + Du[k] \end{aligned}$$

LTI

System Identification: How to create models from data?

Fault Detection and Isolation: How to detect faults from data?

System Identification

Domain Representation

Continuous-time Systems

Discrete-time Systems

$$y^{(n)} + a_{n-1}y^{(n-1)} + \dots + a_0y = b_{m-1}u^{(m-1)} + \dots + b_0u$$

$$y[k] + a_{k-1}y[k-1] + \dots + a_ny[k-n] = b_0u[k] + \dots + b_my[k-m]$$

Laplace Transform

Z Transform

Modern Control

Classical Control

$$(s^n + a_{n-1}s^{n-1} + \dots + a_1s + a_0)Y(s) = (b_{m-1}s^{m-1} + \dots + b_1s + b_0)U(s)$$

$$(1 + a_1z + \dots + a_{n-1}z^{n-1} + a_nz^n)Y(z) = (b_0 + \dots + b_mz^{-m})U(z)$$

$$\begin{aligned} T(s) &= \frac{Y(s)}{U(s)} = \frac{b_ms^m + b_{m-1}s^{m-1} + \dots + b_1s + b_0}{a_ns^n + a_{n-1}s^{n-1} + \dots + a_1s + a_0} \\ &= K \frac{(s-z_1)(s-z_2)\dots(s-z_m)}{(s-p_1)(s-p_2)\dots(s-p_n)} \end{aligned}$$

$$\begin{aligned} T(z) &= \frac{Y(z)}{U(z)} = \frac{b_0 + \dots + b_mz^{-m}}{1 + a_1z + \dots + a_{n-1}z^{n-1} + a_nz^n} \\ &= K \frac{(z-z_1)(z-z_2)\dots(z-z_m)}{(z-p_1)(z-p_2)\dots(z-p_n)} \end{aligned}$$

$$s = j\omega$$

$$z = e^{j\omega T}$$

$$T(j\omega) = |T(j\omega)|e^{\angle T(j\omega)}$$

$$T(e^{j\omega T}) = |T(e^{j\omega T})|e^{\angle T(e^{j\omega T})}$$

Frequency Response

Optimality

Continuous-time Systems

$$y^{(n)} + a_{n-1}y^{(n-1)} + \dots + a_0y = b_{m-1}u^{(m-1)} + \dots + b_0u$$



$$\dot{x} = A(t)x + B(t)u$$

$$y = C(t)x + D(t)u$$

Discrete-time Systems

$$y[k] + a_{k-1}y[k-1] + \dots + a_ny[k-n] = b_ou[k] + \dots + b_mu[k-m]$$



$$x_{k+1} = A_kx_k + B_ku_k$$

$$y_k = C_kx_k + D_ku_k$$

$$\min_{u_k} \frac{1}{2} x_N^T S_N x_N + \frac{1}{2} \sum_{k=0}^{N-1} (x_k^T Q_k x_k + u_k^T R_k u_k)$$

$$\min_{u(t)} \frac{1}{2} x^T(T) S_T x(T) + \frac{1}{2} \int_0^T (x^T(t) Q(t) x(t) + u^T(t) R(t) u(t)) dt$$

Optimal Control

Model Uncertainties

How to deal with uncertainties in the model?

(A) Non-model-based control



PID

Extremum Seeking

(B) Model-based control



1

Robust Control



Design for a family of plants

2

Adaptive Control



Update model (controller) in real time

Robust & Adaptive Control

Model Classification

Model Representation



Control Technique

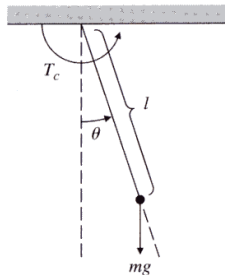
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Dynamic Model

MECHANICAL SYSTEM:



$T = I\alpha$ Newton's law

b damping coefficient

$$I\alpha = -lmg \sin \theta - b\omega + T_c$$

$$\omega = \dot{\theta} \quad \text{angular velocity}$$

$$\alpha = \dot{\omega} = \ddot{\theta} \quad \text{angular acceleration}$$

$$I = ml^2 \quad \text{moment of inertia}$$

$$\ddot{\theta} = -\frac{b}{ml^2} \dot{\theta} - \frac{g}{l} \sin \theta + \frac{T_c}{ml^2}$$

Which are the equilibrium points when $T_c=0$?

At equilibrium: $\ddot{\theta} = \dot{\theta} = 0 \Rightarrow 0 = -\frac{g}{l} \sin \theta \Rightarrow \theta = 0, \pi$

Open loop simulations: pend_par.m, pendol01.mdl

Stable

Unstable

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